CONVECTION HEAT TRANSFER
CONVECTION HEAT TRANSFER
Other books by Adrian Bejan:


The entrepreneur, as a creator of the new and a destroyer of the old, is constantly in conflict with convention. He inhabits a world where belief precedes results, and where the best possibilities are usually invisible to others. His world is dominated by denial, rejection, difficulty, and doubt. And although as an innovator, he is unceasingly imitated when successful, he always remains an outsider to the “establishment.”


In science, the “entrepreneur” is the one who gets the unusual idea, climbs out on a limb, jumps, and runs with it on the landscape. His fate at the feet of the establishment is the same.
## CONTENTS

Preface xv

Preface to the Third Edition xvii

Preface to the Second Edition xxiv

Preface to the First Edition xxvii

List of Symbols xxviii

### 1 Fundamental Principles

1.1 Mass Conservation / 2

1.2 Force Balances (Momentum Equations) / 4

1.3 First Law of Thermodynamics / 8

1.4 Second Law of Thermodynamics / 15

1.5 Rules of Scale Analysis / 17

1.6 Heatlines for Visualizing Convection / 21

References / 22

Problems / 25

### 2 Laminar Boundary Layer Flow

2.1 Fundamental Problem in Convective Heat Transfer / 31

2.2 Concept of Boundary Layer / 34

2.3 Scale Analysis / 37

2.4 Integral Solutions / 42

2.5 Similarity Solutions / 48

2.5.1 Method / 48

2.5.2 Flow Solution / 51

2.5.3 Heat Transfer Solution / 53

2.6 Other Wall Heating Conditions / 56

2.6.1 Unheated Starting Length / 57

2.6.2 Arbitrary Wall Temperature / 58

2.6.3 Uniform Heat Flux / 60

2.6.4 Film Temperature / 61
## CONTENTS

2.7 Longitudinal Pressure Gradient: Flow Past a Wedge and Stagnation Flow / 61
2.8 Flow Through the Wall: Blowing and Suction / 64
2.9 Conduction Across a Solid Coating Deposited on a Wall / 68
2.10 Entropy Generation Minimization in Laminar Boundary Layer Flow / 71
2.11 Heatlines in Laminar Boundary Layer Flow / 74
2.12 Distribution of Heat Sources on a Wall Cooled by Forced Convection / 77
2.13 The Flow of Stresses / 79
References / 80
Problems / 82

### 3 Laminar Duct Flow

3.1 Hydrodynamic Entrance Length / 97
3.2 Fully Developed Flow / 100
3.3 Hydraulic Diameter and Pressure Drop / 103
3.4 Heat Transfer To Fully Developed Duct Flow / 110
   3.4.1 Mean Temperature / 110
   3.4.2 Fully Developed Temperature Profile / 112
   3.4.3 Uniform Wall Heat Flux / 114
   3.4.4 Uniform Wall Temperature / 117
3.5 Heat Transfer to Developing Flow / 120
   3.5.1 Scale Analysis / 121
   3.5.2 Thermally Developing Hagen–Poiseuille Flow / 122
   3.5.3 Thermally and Hydraulically Developing Flow / 128
3.6 Stack of Heat-Generating Plates / 129
3.7 Heatlines in Fully Developed Duct Flow / 134
3.8 Duct Shape for Minimum Flow Resistance / 137
3.9 Tree-Shaped Flow / 139
References / 147
Problems / 153

### 4 External Natural Convection

4.1 Natural Convection as a Heat Engine in Motion / 169
4.2 Laminar Boundary Layer Equations / 173
4.3 Scale Analysis / 176
   4.3.1 High-Pr Fluids / 177
   4.3.2 Low-Pr Fluids / 179
   4.3.3 Observations / 180
4.4 Integral Solution / 182
   4.4.1 High-Pr Fluids / 183
   4.4.2 Low-Pr Fluids / 184
4.5 Similarity Solution / 186
4.6 Uniform Wall Heat Flux / 189
4.7 Effect of Thermal Stratification / 192
4.8 Conjugate Boundary Layers / 195
4.9 Vertical Channel Flow / 197
4.10 Combined Natural and Forced Convection (Mixed Convection) / 200
4.11 Heat Transfer Results Including the Effect of Turbulence / 203
  4.11.1 Vertical Walls / 203
  4.11.2 Inclined Walls / 205
  4.11.3 Horizontal Walls / 207
  4.11.4 Horizontal Cylinder / 209
  4.11.5 Sphere / 209
  4.11.6 Vertical Cylinder / 210
  4.11.7 Other Immersed Bodies / 211
4.12 Stack of Vertical Heat-Generating Plates / 213
4.13 Distribution of Heat Sources on a Vertical Wall / 216
References / 218
Problems / 221

5 Internal Natural Convection 233

5.1 Transient Heating from the Side / 233
  5.1.1 Scale Analysis / 233
  5.1.2 Criterion for Distinct Vertical Layers / 237
  5.1.3 Criterion for Distinct Horizontal Jets / 238
5.2 Boundary Layer Regime / 241
5.3 Shallow Enclosure Limit / 248
5.4 Summary of Results for Heating from the Side / 255
  5.4.1 Isothermal Sidewalls / 255
  5.4.2 Sidewalls with Uniform Heat Flux / 259
  5.4.3 Partially Divided Enclosures / 259
  5.4.4 Triangular Enclosures / 262
5.5 Enclosures Heated from Below / 262
  5.5.1 Heat Transfer Results / 263
  5.5.2 Scale Theory of the Turbulent Regime / 265
  5.5.3 Constructal Theory of Bénard Convection / 267
5.6 Inclined Enclosures / 274
5.7 Annular Space Between Horizontal Cylinders / 276
5.8 Annular Space Between Concentric Spheres / 278
5.9 Enclosures for Thermal Insulation and Mechanical Strength / 278
References / 284
Problems / 289
6 Transition to Turbulence 295

6.1 Empirical Transition Data / 295
6.2 Scaling Laws of Transition / 297
6.3 Buckling of Inviscid Streams / 300
6.4 Local Reynolds Number Criterion for Transition / 304
6.5 Instability of Inviscid Flow / 307
6.6 Transition in Natural Convection on a Vertical Wall / 313
References / 315
Problems / 318

7 Turbulent Boundary Layer Flow 320

7.1 Large-Scale Structure / 320
7.2 Time-Averaged Equations / 322
7.3 Boundary Layer Equations / 325
7.4 Mixing Length Model / 328
7.5 Velocity Distribution / 329
7.6 Wall Friction in Boundary Layer Flow / 336
7.7 Heat Transfer in Boundary Layer Flow / 338
7.8 Theory of Heat Transfer in Turbulent Boundary Layer Flow / 342
7.9 Other External Flows / 347
  7.9.1 Single Cylinder in Cross Flow / 347
  7.9.2 Sphere / 349
  7.9.3 Other Body Shapes / 350
  7.9.4 Arrays of Cylinders in Cross Flow / 351
7.10 Natural Convection Along Vertical Walls / 356
References / 359
Problems / 361

8 Turbulent Duct Flow 369

8.1 Velocity Distribution / 369
8.2 Friction Factor and Pressure Drop / 371
8.3 Heat Transfer Coefficient / 376
8.4 Total Heat Transfer Rate / 380
  8.4.1 Isothermal Wall / 380
  8.4.2 Uniform Wall Heating / 382
  8.4.3 Time-Dependent Heat Transfer / 382
8.5 More Refined Turbulence Models / 383
8.6 Heatlines in Turbulent Flow Near a Wall / 387
8.7 Channel Spacings for Turbulent Flow / 389
References / 390
Problems / 392
9 Free Turbulent Flows

9.1 Free Shear Layers / 398
  9.1.1 Free Turbulent Flow Model / 398
  9.1.2 Velocity Distribution / 401
  9.1.3 Structure of Free Turbulent Flows / 402
  9.1.4 Temperature Distribution / 404

9.2 Jets / 405
  9.2.1 Two-Dimensional Jets / 406
  9.2.2 Round Jets / 409
  9.2.3 Jet in Density-Stratified Reservoir / 411

9.3 Plumes / 413
  9.3.1 Round Plume and the Entrainment Hypothesis / 413
  9.3.2 Pulsating Frequency of Pool Fires / 418
  9.3.3 Geometric Similarity of Free Turbulent Flows / 421

9.4 Thermal Wakes Behind Concentrated Sources / 422

References / 425
Problems / 426

10 Convection with Change of Phase

10.1 Condensation / 428
  10.1.1 Laminar Film on a Vertical Surface / 428
  10.1.2 Turbulent Film on a Vertical Surface / 435
  10.1.3 Film Condensation in Other Configurations / 438
  10.1.4 Drop Condensation / 445

10.2 Boiling / 447
  10.2.1 Pool Boiling Regimes / 447
  10.2.2 Nucleate Boiling and Peak Heat Flux / 451
  10.2.3 Film Boiling and Minimum Heat Flux / 454
  10.2.4 Flow Boiling / 457

10.3 Contact Melting and Lubrication / 457
  10.3.1 Plane Surfaces with Relative Motion / 458
  10.3.2 Other Contact Melting Configurations / 462
  10.3.3 Scale Analysis and Correlation / 464
  10.3.4 Melting Due to Viscous Heating in the Liquid Film / 466

10.4 Melting By Natural Convection / 469
  10.4.1 Transition from the Conduction Regime to the Convection Regime / 469
  10.4.2 Quasisteady Convection Regime / 472
  10.4.3 Horizontal Spreading of the Melt Layer / 474

References / 478
Problems / 482
11 Mass Transfer 489

11.1 Properties of Mixtures / 489
11.2 Mass Conservation / 492
11.3 Mass Diffusivities / 497
11.4 Boundary Conditions / 499
11.5 Laminar Forced Convection / 501
11.6 Impermeable Surface Model / 504
11.7 Other External Forced Convection Configurations / 506
11.8 Internal Forced Convection / 509
11.9 Natural Convection / 511
   11.9.1 Mass-Transfer-Driven Flow / 512
   11.9.2 Heat-Transfer-Driven Flow / 513
11.10 Turbulent Flow / 516
   11.10.1 Time-Averaged Concentration Equation / 516
   11.10.2 Forced Convection Results / 517
   11.10.3 Contaminant Removal from a Ventilated Enclosure / 520
11.11 Massfunction and Masslines / 527
11.12 Effect of Chemical Reaction / 527
References / 531
Problems / 532

12 Convection in Porous Media 537

12.1 Mass Conservation / 537
12.2 Darcy Flow Model and the Forchheimer Modification / 540
12.3 First Law of Thermodynamics / 542
12.4 Second Law of Thermodynamics / 546
12.5 Forced Convection / 547
   12.5.1 Boundary Layers / 547
   12.5.2 Concentrated Heat Sources / 552
   12.5.3 Sphere and Cylinder in Cross Flow / 553
   12.5.4 Channel Filled with Porous Medium / 554
12.6 Natural Convection Boundary Layers / 555
   12.6.1 Boundary Layer Equations: Vertical Wall / 555
   12.6.2 Uniform Wall Temperature / 556
   12.6.3 Uniform Wall Heat Flux / 558
   12.6.4 Spacings for Channels Filled with Porous Structures / 559
   12.6.5 Conjugate Boundary Layers / 562
   12.6.6 Thermal Stratification / 563
   12.6.7 Sphere and Horizontal Cylinder / 566
   12.6.8 Horizontal Walls / 567
   12.6.9 Concentrated Heat Sources / 567
PREFACE

An author is fortunate if his book is popular enough to merit a second edition somewhere down the line, yet the flow of ideas that grew around this book since the first edition (1988) has been beyond expectations. I will let others comment on this flow. In this brief Preface, I comment on just one feature of the flow of ideas and one bit of history.

The flow of ideas is illustrated by the changes made in this new edition. Good ideas (in this or any other field) attract interesting minds—researchers, educators, and authors with ideas. These minds grow the field the way that the yeast grows the cake. While revising this edition, it was not possible to keep up with this growth, but I tried, even though this meant abandoning some of the material from earlier editions. The new growth is represented by the impact of the science of discovering effective flow configurations (constructal theory and design), the streamlining of the discipline along methods that are direct, muscular, and at the same time lean (scale analysis, intersection of asymptotes, heatlines), the oneness with thermodynamics through the irreversibility (entropy generation) phenomenon, and new references and problems at the end of chapters.

Because we know where convection and thermodynamics come from, this growth illustrates that science (education, knowledge, information) is an evolutionary design [1–4], a flow system that constantly morphs and improves so that our own movement and life are facilitated and extended on the landscape. This is nature, the animate and the inanimate alike.

Because research is autobiographical, good research is a book of wonderful memories. I close this preface with the story of how the first edition of this book was born. It was an accident, literally. At age 33, I was behaving as if I was meant to play basketball forever, and I was wrong. During a game in January 1982, one of my Achilles’ tendons was severed, and I ended up in a wheelchair for the entire semester. I had to teach my convection course, for which I had written notes, but this time I was forced to write each lecture on transparencies, for the screen. My first graduate student, Shigeo Kimura, now professor at Kanazawa University, Japan, was my teaching assistant. He would wheel me into the classroom every morning, and my convection book would come to life, one original drawing at
a time, one original (solved) problem after another. One such problem was the method of intersecting the asymptotes and the back-of-the-envelope prediction of optimal spacings (Problem 11, Chapter 4, p. 157, in the first edition).

There was so much richness during the spring of 1982 that the accident was a blessing.

ADRIAN BEJAN

Duke University

REFERENCES

Research is autobiographical. I often say this when I lecture, and I find it true as I look at this new edition of *Convection Heat Transfer*. It is even more true as I look at all three editions together. This book is a chronicle of the heat transfer side of my career, the methods I developed and taught along the way, and the great fortune I had to work with extremely gifted colleagues. The three editions are also a story of how the field has grown and prospered. It has done so based on new challenges and especially, new ideas.

One trend that is made visible (and useful, I hope) in this edition is the new emphasis on *design as science*—the generation of flow configuration based on principle. For many years, the field of convection was preoccupied with documenting the transport characteristics of various but simple flow configurations—relationships between temperature differences and heat transfer rates. This information is essential in the modeling and simulations that are necessary in design. The reality, however, is harsh: Constraints exist, and one overriding constraint is space (size, volume, weight). Putting more and more heat transfer into a given volume has been the objective, from the compact heat exchangers of my MIT years to the heat transfer augmentation techniques and the cooling of electronics packages of today. Doing more with limited resources has been the driving force.

Miniaturization marches forward, but this is not even half of the story. The reason is that the devices we touch must be made at our scale—they must be macroscopic, no matter how small the smallest components. The more successful we are in making smaller components, the greater the challenge to install larger numbers of such components and to connect them with currents (heat, fluid, electricity), to keep them alive. The challenge is to “construct,” to assemble and design while assembling (i.e., to design complexity and to deduce the flow configuration of the macroscopic device).

*Construction* must be shouted from the rooftops, especially today as the crowd marches toward smaller scales. To construct is to proceed in the opposite
direction, from small to large, because only in this direction can the small scales be made useful. Only after the achievement of constructal assembly can small-scale components deliver high densities of heat transfer.

In this new edition, the first steps toward constructs with high heat transfer density are used as an introduction to constructal theory and design*: the generation of flow architecture in the pursuit of maximal global performance subject to global constraints, when the flow architecture is free to morph. The focus is on method, on design as science, on the generation of optimal and complex architectures based on the constructal law. To emphasize this facet of the third edition is appropriate not only because of its importance today, but also because it had its start in the 1984 edition [see the optimization of spacings with natural convection (p. 157, Problem 11, Chapter 4).

The focus on methodology is why in this new edition I chart the progress made by three other methods that were pioneered in the 1984 edition. These methods have become recognized and now occupy growing sections of the literature:

The intersection of asymptotes method, which delivered in amazingly direct fashion the optimal spacing for natural convection (see above), has since been extended to spacings for forced convection and the constructal theory prediction of all the basic features of Bénard convection. The intersection of asymptotes is also useful pedagogically, in the teaching of the concept of transition (e.g., laminar–turbulent flow, natural–forced convection).

Heatlines are now being used to visualize the true paths followed by convection: the paths of energy flow, not fluid flow. They were introduced in the 1984 edition, with an example of natural convection in an enclosure. The concept has since been extended to mass transfer and a variety of basic and applied configurations with natural and forced convection in fluids and fluid-saturated porous media. This method of visualization is particularly well suited for computational heat transfer and should be included in commercial computational packages.

Scale analysis continues to be the main method for teaching the basics of convection in this new edition. The rules and promise of scale analysis as a problem-solving method were first formulated in the 1984 edition. Today the method is used widely, and this makes it even more essential in a basic course of convection. The increased importance of scale analysis is also due to the proliferation of computational heat transfer. If done correctly, scale analysis can shed light on what the deluge of numerical results is trying to tell us. Even more, to teach scale analysis is to remind the student not to give up on pencil and paper. Not everything must be done on the computer.

Porous media were brought into a heat transfer course for the first time by the 1984 edition of this book. Since then, convection in porous media has developed into a field of its own. In this edition we continue to emphasize the basic method and the most basic results. A connection is also made between porous media and

designed complex flow structures,* and this serves as one more bridge to the constructal design method.

*Interdisciplinary teaching and research* is one of the missions of this course, but with this warning: Learn your disciplines first; only then you will be strong on the interdisciplinary frontiers. The teaching of convection in porous media is a good example. This is presented not as a self-standing subject but as an interaction between principles of convection in pure fluids, which we all learn, and newly emerging technological applications that employ porous flow structures.


ADRIAN BEJAN

*Durham, North Carolina*

*April 2004*

I want to thank John Wiley & Sons, Inc. and the users of my *Convection Heat Transfer* for giving me this opportunity to prepare a second edition. The changes and additions that I made are due to the suggestions received from many colleagues and students, and to the evolution of my own research activity.

I made changes in both format and content. The format is now based on numbered sections and equations, to make it easier for the first-time user to use this book as a reference. I assembled all the symbols in a list that precedes the text. The Author Index acknowledges one more time the individuals whose work is quoted in the text. The Solutions Manual is now produced on the word processor, and has the appearance of a companion book.

The changes in content are more significant and at more than one level. New topics covered in the second edition are convection with change of phase (condensation, boiling, melting), the cooling of electronic packages by forced and natural convection, lubrication by contact melting, and several examples of conjugate heat transfer, i.e., convection coupled with conduction or radiation. I augmented most chapters with results, namely, formulas, tables, charts, and appendixes that are recommended for use in engineering design work. And, speaking of design, many of the new problems at the end of chapters refer to basic principles of thermal design.

Relative to the first edition, the chapters dealing with laminar and, especially, turbulent forced convection have been expanded. To make room for the new material and still respect the prescribed space limits, I had to eliminate the chapter on numerical methods, and to condense the treatment of convection in porous media. Numerical methods are now covered in courses devoted entirely to computational fluid dynamics and heat transfer. For porous media, I recently completed with Professor D. A. Nield a separate textbook, *Convection in Porous Media* (Springer, 1992; now in 4th edition, 2013).
As in the first edition, the most important feature of this book is that many of the topics and problems came from my own research. These problems recommended themselves as interesting and beautiful, i.e., worthy of study. They represent my argument in favor of practicing *laissez faire* in engineering research, and against the *dirigiste* policy advocated by others.

*Durham, North Carolina*
*June 1994*

Adrian Bejan
PREFACE TO THE FIRST EDITION

My main reason for writing a convection textbook is to place the field’s past 100 years of growth in perspective. This book is intended for the educator who wants to present his students with more than a review of the generally accepted “classical” methods and conclusions. Through this book I hope to encourage the convection student to question what is known and to think freely and creatively about what is unknown.

There is no such thing as “unanimous agreement” on any topic. The history of scientific progress shows clearly that our present knowledge and understanding—contents of today’s textbooks—are the direct result of conflict and controversy. By encouraging our students to question authority, we encourage them to make discoveries on their own. We can all only benefit from the scientific progress that results.

In writing this book, I sought to make available a textbook alternative that offers something new on two other fronts: (1) content, or the selection of topics, and (2) method, or the approach to solving problems in convection heat transfer.

Regarding content, this textbook reflects the relative change in the priorities set by our technological society over the past two decades. Historically, the field of convective heat transfer grew out of great engineering pursuits such as energy conversion (power plant technology), the aircraft, and the exploration of extraterrestrial space. Today, we are forced to face additional challenges, primarily in the areas of “energy” and “ecology.” Briefly stated, engineering education today places a strong emphasis on man’s need to coexist with the environment. This new emphasis is reflected in the topics assembled in this book. Important areas covered for the first time in a convection textbook are: (1) natural convection on an equal footing with forced convection, with application to energy conservation in buildings and to geophysical dynamics, (2) convection through porous media saturated with fluid, with application to geothermal and thermal insulation engineering, and (3) turbulent mixing in free-stream flow, with application to the dispersion of pollutants in the atmosphere and the hydrosphere.
Regarding method, in this book I made a consistent effort to teach problem solving (a Solutions Manual is available from the publisher or from me). This book is a textbook to be used for teaching a course, not a handbook. Of course, important engineering results are listed; however, the emphasis is placed on the thinking that leads to these results. A unique feature of this book is that it stresses the importance of correct scale analysis as an eligible and cost-effective method of solution, and as a precondition for more refined methods of solution. It also stresses the need for correct scaling in the graphic reporting of more refined analytical results and of experimental and numerical data. The cost and the ‘‘return on investment’’ associated with a possible method of solution are issues that each student-researcher should examine critically: these issues are stressed throughout the text.

I wrote this book during the academic year 1982–1983, in our mountain-side house on the greenbelt of North Boulder. This project turned out to be a highly rewarding intellectual experience for me, because it forced upon me the rare opportunity to think about an entire field, while continuing my own research on special topics in convection and other areas (specialization usually inhibits the ability to enjoy a bird’s-eye-view of anything). It is a cliché in education and research for the author of a new book to end the preface by thanking his family for the ‘‘sacrifice’’ that allowed completion of the work. My experience with writing Convection Heat Transfer has been totally different (i.e., much more enjoyable!), to the point that I must thank this book for making me work at home and for triggering so many inspiring conversations with Mary. Convection can be entertaining.

ADRIAN BEJAN

Boulder, Colorado
July 1984
**LIST OF SYMBOLS**

<table>
<thead>
<tr>
<th>Symbol</th>
<th>Description</th>
</tr>
</thead>
<tbody>
<tr>
<td>(a, b)</td>
<td>dimensions of rectangular duct cross section (Fig. 3.5)</td>
</tr>
<tr>
<td>(A)</td>
<td>area</td>
</tr>
<tr>
<td>(A_c)</td>
<td>cross-sectional area</td>
</tr>
<tr>
<td>(A, B)</td>
<td>constants in the logarithmic law of the wall [eqs. (7.41) and (7.42)]</td>
</tr>
<tr>
<td>(Ar)</td>
<td>Archimedes number [eq. (10.80)]</td>
</tr>
<tr>
<td>(b)</td>
<td>empirical constant, Forchheimer flow [eq. (12.15)]</td>
</tr>
<tr>
<td>(b)</td>
<td>natural convection parameter [eq. (5.117)]</td>
</tr>
<tr>
<td>(b)</td>
<td>radial length scale of round velocity jet [eq. (9.40)]</td>
</tr>
<tr>
<td>(b)</td>
<td>stratification parameter [eq. (12.116)]</td>
</tr>
<tr>
<td>(b)</td>
<td>taper parameter [eq. (2.140)]</td>
</tr>
<tr>
<td>(b)</td>
<td>thermal stratification number [eq. (4.81)]</td>
</tr>
<tr>
<td>(b_r)</td>
<td>radial length scale of round thermal jet [eq. (9.43)]</td>
</tr>
<tr>
<td>(\tilde{b}_{1,2})</td>
<td>empirical factors (Table 11.6)</td>
</tr>
<tr>
<td>(B)</td>
<td>condensation driving parameter [eq. (10.26)]</td>
</tr>
<tr>
<td>(B)</td>
<td>cross-sectional shape number (Fig. 3.7)</td>
</tr>
<tr>
<td>(B)</td>
<td>dimensionless group [eq. (2.147)]</td>
</tr>
<tr>
<td>(B)</td>
<td>dimensionless group [eq. (12.107)]</td>
</tr>
<tr>
<td>(Be_L)</td>
<td>Bejan number, pressure drop number [eq. (3.120')]</td>
</tr>
<tr>
<td>(Be_p)</td>
<td>Bejan number for a porous medium [eq. (12.113)]</td>
</tr>
<tr>
<td>(Bo_H)</td>
<td>Boussinesq number [eq. (4.35)]</td>
</tr>
<tr>
<td>(c)</td>
<td>specific heat of incompressible substance</td>
</tr>
<tr>
<td>(c_v)</td>
<td>specific heat at constant volume</td>
</tr>
<tr>
<td>(c_p)</td>
<td>specific heat at constant pressure</td>
</tr>
<tr>
<td>(c_{1,2})</td>
<td>constants</td>
</tr>
<tr>
<td>(C)</td>
<td>compressive impulse or reaction [eq. (6.7)]</td>
</tr>
<tr>
<td>(C)</td>
<td>concentration [eq. (11.1)]</td>
</tr>
<tr>
<td>(C)</td>
<td>constant</td>
</tr>
<tr>
<td>(C_{f,x})</td>
<td>local skin friction coefficient [eqs. (2.57) and (7.52)]</td>
</tr>
<tr>
<td>(C_n)</td>
<td>factor (Fig. 7.11)</td>
</tr>
<tr>
<td>(C_1, C_2, C_\mu)</td>
<td>constants [eq. (8.61)]</td>
</tr>
<tr>
<td>(C_D)</td>
<td>drag coefficient [eq. (7.103)]</td>
</tr>
<tr>
<td>(C_{sf})</td>
<td>constant (Table 10.1)</td>
</tr>
<tr>
<td>(d, D)</td>
<td>diameter</td>
</tr>
<tr>
<td>(D)</td>
<td>mass diffusivity [eq. (11.24), Tables 11.1 and 11.2]</td>
</tr>
<tr>
<td>(D)</td>
<td>plate-to-plate spacing (Fig. 3.1)</td>
</tr>
<tr>
<td>(D)</td>
<td>stream transversal length scale</td>
</tr>
<tr>
<td>(D_h)</td>
<td>hydraulic diameter [eq. (3.26)]</td>
</tr>
<tr>
<td>(D_{k-k})</td>
<td>knee-to-knee thickness of time-averaged turbulent shear layer (Fig. 9.3)</td>
</tr>
</tbody>
</table>
\( D_T \) distance of maximum thermal penetration in the y direction, in the vicinity of a direct contact spot [eq. (7.94)]

\( e \) specific energy (labeled \( u \) in Table 1.1)

\( f \) Blasius streamfunction similarity profile [eq. (2.80)]

\( f \) factor [eq. (7.113)]

\( f \) friction factor [eq. (3.24)]

\( f \) porous medium friction factor [eq. (12.12)]

\( f \) roll thickness [eq. (5.92)]

\( f_u \) curve fit for the velocity profile [eq. (7.53)]

\( f_v \) frequency of vortex shedding [eq. (7.102)]

\( F \) force

\( F \) streamfunction similarity profile [eqs. (4.60) and (12.139)]

\( F_0 \) Fourier number [eq. (10.104)]

\( F_D \) drag force

\( F_n \) normal force

\( F_t \) tangential force

\( g \) gravitational acceleration

\( \text{Gr}_H \) Grashof number [eq. (4.38)]

\( \text{Gr}_* \) Grashof number based on heat flux (Table 6.1)

\( \text{Gz} \) Graetz number [eq. (3.107)]

\( \xi \) constant (Table 4.3)

\( h \) heat transfer coefficient [eq. (2.4)]; local heat transfer coefficient [eq. (2.100)]

\( h \) specific enthalpy

\( h_{fg} \) latent heat of condensation or evaporation (Table 10.2)

\( h'_{fg} \) augmented latent heat [eq. (10.10)]

\( h''_{fg} \) augmented latent heat [eq. (10.41)]

\( h_m \) mass transfer coefficient [eq. (11.46)]

\( h_{sf} \) latent heat of melting

\( H \) enthalpy flow rate [eq. (10.5)]

\( H \) heatfunction [defined via eqs. (1.68) and (1.69)]

\( H \) height

\( H \) Henry’s constant [eq. (11.35) and Table 11.3]

\( I \) area moment of inertia

\( I \) integral [eq. (3.135)]

\( j \) diffusion flux [eq. (11.20)]

\( j_{app} \) apparent mass flux [eq. (11.102)]

\( J \) dimensionless thickness parameter [eq. (2.139)]

\( \text{Ja} \) Jakob number [eq. (10.19)]

\( k \) thermal conductivity

\( k \) wave number

\( k''_n, k'''_n \) reaction rates [eqs. (11.135) and (11.136)]

\( k_s \) sand grain size [eq. (8.16)]
\( K \) jet strength [eq. (9.33)]
\( K \) permeability [eq. (12.9)]
\( K_{1,2} \) constants
\( l \) effective length [eq. (4.127)]
\( l \) mixing length [eq. (7.27)]
\( L \) length
\( L \) length of direct viscous contact [eq. (7.92)]
\( L_c \) characteristic length
\( \ell \) equivalent length [eq. (10.86)]
\( L_{m\text{c}} \) length of direct thermal contact [eq. (7.95)]
\( \ell_c \) effective length [eq. (4.128)]
\( \text{Le} \) Lewis number [eq. (11.93)]
\( m \) exponent in flow over a wedge [eq. (2.124)]
\( m \) function [eq. (6.27)]
\( m \) profile shape function for integral analysis [eq. (2.54)]
\( \dot{m} \) mass flow rate
\( \dot{m}' \) mass transfer rate per unit length [eq. (11.52)]
\( \dot{m}'' \) volumetric mass generation rate [eq. (11.15)]
\( M \) bending moment [eq. (6.8)]
\( M \) function [eq. (8.22)]
\( M \) impulse or reaction force due to fluid flow into or out of a control volume (Fig. 2.3)
\( M \) mass
\( M \) massfunction [eqs. (11.133)-(11.134)]
\( M \) material constraint [eq. (3.132)]
\( M \) molar mass [eq. (11.4)]
\( n \) dimensionless coordinate across the velocity boundary layer \((y/\delta)\) [eq. (2.54)]
\( n \) number of cylinders
\( n \) number of heat-generating boards
\( n \) number of moles [eq. (11.4)]
\( n_t \) number of rows
\( N_B \) buckling number [eq. (6.14)]
\( N_{htu} \) number of heat transfer units [eq. (8.56)]
\( \text{Nu} \) local Nusselt number [eq. (2.101)]
\( \text{Nu} \) Nusselt number in the fully developed region [eq. (3.52)]
\( \text{Nu}_0 \) overall Nusselt number
\( \text{Nu}_0 \) constant (Table 4.3)
\( \text{Nu}_{0-x} \) overall Nusselt number [eq. (3.91)]
\( \text{Nu}_x \) local Nusselt number in the developing (entrance) region [eq. (3.90)]
\( p \) dimensionless coordinate across the thermal boundary layer \((y/\delta_T)\) [eq. (2.58)]
\( p \) even function (eq. (5.37))
List of Symbols

- $p$: wetted perimeter
- $P$: pressure
- $P_\infty$: pressure in the free stream
- $P_{\infty}$: pressure in the free stream
- $Pe_D$: Péclet number ($UD/\alpha$)
- $Pe_L$: Péclet number ($U_\infty L/\alpha$)
- $Po$: Poiseuille number ($f Re_{Dh}$)
- $Pr$: Prandtl number ($\nu/\alpha$)
  - $Pr_p$: porous medium Prandtl number [eq. (12.215)]
  - $Pr_t$: turbulent Prandtl number [eq. (7.66)]
- $q$: heat transfer rate (W)
- $q^\prime$: heat transfer rate per unit length (W/m)
- $q^\prime\prime$: heat flux (W/m²)
  - $q^\prime\prime_{\text{app}}$: apparent heat flux [eq. (7.24)]
- $q^\prime\prime_{0, \text{max}}$: maximum heat flux, under a direct thermal contact spot [eq. (7.86)]
- $q^\prime\prime\prime$: rate of internal heat generation (W/m³)
- $Q$: heat transfer rate (W)
- $Q$: flow rate ($m²/s$) [eq. (10.69)]
- $r$: radial coordinate
- $r_0$: tube radius
- $r_h$: hydraulic radius [eq. (3.26)]
- $r$, $\theta$, $z$: cylindrical coordinates (Fig. 1.1)
- $r$, $\phi$, $\theta$: spherical coordinates (Fig. 1.1)
- $R$: ideal gas constant
- $R$: universal gas constant
- $R$: radius
- $R$: thermal resistance
- $Ra_H$: Rayleigh number [eq. (4.25)]
- $Ra_y$: Darcy modified Rayleigh number [eq. (12.89)]
- $Ra_{m,y}$: mass transfer Rayleigh number [eq. (11.86)]
- $Ra_q$: Rayleigh number based on source strength [eq. (6.6)]
- $Ra_{Hq}$: Rayleigh number based on heat flux [eq. (4.70)]
- $Ra_{yq}$: Darcy modified Rayleigh number based on heat flux [eq. (12.99)]
- $Re_D$: Reynolds number ($UD/\nu$)
- $Re_{Dh}$: Reynolds number based on hydraulic diameter ($UD_h/\nu$)
- $Re_l$: local Reynolds number [eq. (6.15)]
- $Re_L$: Reynolds number ($U_\infty L/\nu$)
- $Re_t$: terminal Reynolds number [eq. (10.37)]
- $s$: constant (Table 10.1)
- $s$: specific entropy
- $s$: thickness of liquid zone (Fig. 10.24)
- $S$: entropy (J/K)